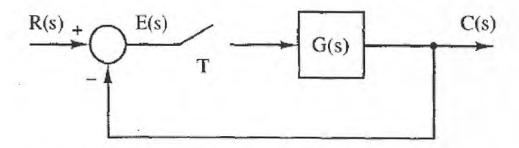
Chapter 4

4.1 STEADY-STATE ACCURACY

An important characteristic of a control system is its ability to follow, or track, certain inputs with a minimum of error. The control system designer attempts to minimize the system error to certain anticipated inputs. In this section the effects of the system transfer characteristics on the steady-state system errors are considered.

Consider the system of Figure



$$\frac{C(z)}{R(z)} = \frac{G(z)}{1 + G(z)}$$

the system error, e(t), is defined as

$$E(z) = \mathfrak{z}[e(t)] = R(z) - C(z)$$

$$E(z) = R(z) - \frac{G(z)}{1 + G(z)}R(z) = \frac{R(z)}{1 + G(z)}$$

The steady-state errors will now be derived for two common inputs—a position (step) input and a velocity (ramp) input. First, for the unit-step input,

$$R(z) = \frac{z}{z - 1}$$

provided that $e_{ss}(kT)$ has a final value. The steady-state error is then

$$e_{ss}(kT) = \lim_{z \to 1} \frac{z}{1 + G(z)} = \frac{1}{1 + \lim_{z \to 1} G(z)}$$

We now define the position error constant as

$$K_p = \lim_{z \to 1} G(z)$$

$$e_{\rm ss}(kT) = \frac{1}{1+K_p}$$

(system type greater than or equal to one), $K_p = \infty$ and the steady-state error is zero.

Consider next the unit-ramp input. In this case r(t) = t

$$R(z) = \frac{Tz}{(z-1)^2}$$

$$e_{ss}(kT) = \lim_{z \to 1} \frac{Tz}{(z-1) + (z-1)G(z)} = \frac{T}{\lim_{z \to 1} (z-1)G(z)}$$

We now define the velocity error constant as

$$K_{\nu} = \lim_{z \to 1} \frac{1}{T}(z-1)G(z)$$

$$e_{\rm ss}(kT)=\frac{1}{K_{\nu}}$$

(system type greater than or equal to 2), $K_{\nu} = \infty$ and $e_{ss}(kT)$ is zero.

The development above illustrates that, in general, increased system gain and/or the addition of poles at z = 1 to the open-loop forward-path transfer function

tend to decrease steady-state errors.

both large gains and poles of G(z) at z = 1 have destabilizing effects on the system. Generally, trade-offs exist between small steady-state errors and adequate system stability (or acceptable system transient response).

Example

The steady-state errors will be calculated for the system of Figure open-loop function is given as

$$G(s) = \frac{1 - e^{-Ts}}{s} \left[\frac{K}{s(s+1)} \right]$$

Thus

$$G(z) = K_{\mathcal{J}} \left[\frac{1 - \epsilon^{-T_{s}}}{s^{2}(s+1)} \right] = \frac{K(z-1)}{z} \mathcal{J} \left[\frac{1}{s^{2}(s+1)} \right]$$

$$= \frac{K(z-1)}{z} \frac{z[(\epsilon^{-T} + T - 1)z + (1 - \epsilon^{-T} - T\epsilon^{-T})]}{(z-1)^{2}(z - \epsilon^{-T})}$$

$$= \frac{K[(\epsilon^{-T} + T - 1)z + (1 - \epsilon^{-T} - T\epsilon^{-T})]}{(z-1)(z - \epsilon^{-T})}$$

the system is type 1 and

Kv= K

Since G(z) has one pole at z = 1, the steady-state error to a step input is zero, and to

a ramp input is,

$$e_{\rm ss}(kT)=\frac{1}{K_{\rm v}}=\frac{1}{K}$$

Note that

(z-1) is represent a pole at z=1 or (Type one system)

$$(z-1)^2$$
 is represent a 2-pole at z=1 or (Type two system)

Example:

The open-loop T.F is:-

$$G(z) = \frac{0.98z + 0.66}{(z - 1)(z - 0.368)}$$

Compute Constants and Ess.

Sol

$$Kp = \lim_{z \to 1} \frac{0.98z + 0.66}{(z - 1)(z - 0.368)}$$

$$Kp = \infty$$

$$Ess = \frac{1}{1 + Kp}$$

$$Ess = 0$$

$$Kv = \frac{1}{T} \lim_{z \to 1} [(z-1) \frac{0.98z + 0.66}{(z-1)(z-0.368)}$$

$$Kv = \frac{2.59}{T}$$

$$Ess = \frac{1}{Kv} = \frac{T}{2.59}$$

Note that

In sometimes the (Ess) can be used as one of the design requirements in control system.

Example

Let the O.L.T.F is:-

$$G(s) = \frac{K}{s(s+1)}$$

Calculate the gain(K) for Ess=1%. Suppose the i/p signal ramp and T=0.1sec

Sol

$$G(z) = (1 - z^{-1}) \frac{G(s)}{s}$$

$$G(z) = \frac{(0.005z + 0.00347)K}{(z - 1)(z - 0.9048)}$$

$$Kv = \frac{1}{T} \lim_{z \to \infty} [(z - 1)G(z)]$$

$$Kv = 0.918K$$

$$Ess = \frac{1}{Kv} = \frac{1}{0.918K} = 0.01$$

$$K = 108.93$$

NOTES

1-May be gives in question Ess large than or small a tolerance band.

And calculate (K) for this band.

2- May be there is a controller D(z) in feed forward path with a system G(z). so that the O.L.T.F is become G(z)D(z) instead of G(z) alone.

In general

Steady State Error and System Type

System	Steady-state errors in response to		
	Step input $r(t) = 1$	Ramp input $r(t) = t$	Acceleration input $r(t) = \frac{1}{2}t^2$
Type 0 system	$\frac{1}{1+K_p}$	×	∞.
Type 1 system	0	1 K	Œ
Type 2 system	0	0	$\frac{1}{K}$

4.2 How to calculate ζ, Wn for discrete control system

Suppose, you have a 2nd order continuous control system

if
$$\frac{C(s)}{R(s)} = \frac{Wn^2}{s^2 + 2\zeta Wns + Wn^2}$$

Which has the poles:-

Thus given the complex pole locations in the Z-domain, we find (ζ, Wn, τ) by using equations (5,6,7).

Example:

$$\frac{C(z)}{R(z)} = \frac{0.368z + 0.264}{z^2 - z + 0.632}$$

$$z^{2} - z + 0.632 = z^{2} - 2a + a^{2} + b^{2}$$

 $a = 0.5$
 $b = 0.618$
 $polesZ_{1,2} = 0.5 + 0.618j$
 $R = \sqrt{a^{2} + b^{2}}$(*)
 $\theta = \tan^{-1} \frac{b}{a}$(**)
 $R = 0.795$
 $\theta = 51 = 0.89rad$
 $u \sin g(5,6,7)$, We calculate 1- $\zeta = 0.25$
 $Wn = 0.919rad / \sec \tau = 4.36 \sec$

To analyze the same example in continuous system

The original system is

$$G(s) = \frac{1}{s^2 + s + 1}$$

$$\zeta = 0.5$$

$$Wn = \frac{1}{s^2 + s + 1}$$

$$\zeta = 0.5$$

$$V = \frac{1}{s^2 + s + 1}$$

Note that

The effects of the sampling (T=1 sec) is seem to destabilizing, however, if (T=0.1sec) instead of (T=1sec in above example), there is little effect from sampling.

Example:

If the c/cs equation is

$$Q(z) = z^2 - 1.9z + 0.91$$

Find ζ, Wn, τ

Using eq(5,6,7), we get:-

 $\zeta = 0.46$

Wn=1.0245rad/sec

 $\tau = 2.12 \operatorname{sec}$

4.3 How to calculate G(s) from the discrete control system G(z)

Example:

Let, G(z)

$$G(z) = \frac{C(z)}{R(z)} = \frac{0.2909z + 0.1693}{z^2 + 0.7417z + 0.2019}$$

$$T = 0.2 \operatorname{sec}$$

Find G(s)

Sol:-

You know

$$\frac{C(s)}{R(s)} = \frac{Wn^2}{s^2 + 2\zeta Wns + Wn^2} = \frac{wn^2}{(s + \sigma + Wdj)(s + \sigma - Wdj)}$$

$$S_{1,2} = -\zeta w n^{\frac{1}{2}} jw n \sqrt{1 - \zeta^2}$$

$$S_{1,2} = -\sigma + jWd$$

$$\theta/T = Wd....(1) \rightarrow \text{Fwm equ. 2 } \rho.112$$

 $-\sigma = \ln R/T...(2) \rightarrow \text{From equ. 3 } \rho.112$

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Calculate φ and R using equations(*) and(**) respectively

R=0.4494

Φ=34.372=2.5417rad

To calculate (σ) and (Wd) and sub. In the standard 2^{nd} order equation G(s).

Note that

In sometimes, you have ζ and Wn for digital control system, how to calculate the c/cs equation Q(z)

Use equ.(5) and (6) to calculate (R) and (φ). Where:-

$$R = \sqrt{a^2 + b^2}$$

$$\theta = \tan^{-1} \frac{b}{a}$$

From these equations, we get (a) and (b)

$$Q(z) = (z - a + bj)(z - a - bj)$$

Example:

If ζ =0.25 and Wn=0.919rad/sec and T=1 sec.

R=0.795

 $\phi = 0.89$

a=0.5 and b=0.6177

$$Q(z) = z^2 - z + 0.632$$

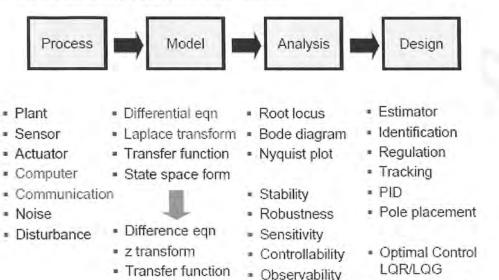
Chapter 5

Design of digital control system

Adaptive control

Robust control

. The Research Procedure in Control Science



5.1 Design of Discrete-Time Controllers

State space form

Two approaches are possible to the design of digital control laws:

1. "Direct" method

- Discretization of the plant model
- Design of the controller in the discrete-time domain

2. "Indirect" Method

- Simplest approach, it does not requires specific knowledge of design techniques in the discrete-time domain
- Some limitations are given by the choice of the sampling time

